Hydroflouroolefins (HFOs) as Low GWP Refrigerants for Residential Heat Pump Water Heaters

Kashif Nawaz, Bo Shen, Ahmed Elatar, Van Baxter

Building Technologies Research and Integration Center, ORNL

nawazk@ornl.gov

ACEEE Hot Water Forum 2017





ORNL is managed by UT-Battelle for the US Department of Energy

Content

- Background
- Potential alternative refrigerants
- ORNL Heat Pump Design Model (HPDM)
- Performance comparisons
- Summary



Acknowledgements

- DOE Building Technologies Office, Emerging Technologies – Antonio Bouza
- Dr. Omar Abdelaziz (ORNL)



Consequences of Global Warming

- Glacier retreat
- Sever weather
- Droughts and large scale fires



Naska's Pedersen Glacier as it looked in the 1920s 1940s (left) and as it looked when photographed in 2005 (right). About 90 percent of the world's glaciers are shrinking, and their reduction in size worldwide has accelerated rapidly since the 1970s, according to the Worldwide Glacier Monitoring Service. (Photo by USGS/Bruce Molnia)

The warnings about global warming have been extremely clear for a long time. We are facing a global climate crisis. It is deepening, We are entering a period of consequences.





Next Generation Refrigerants

Refrigerant	GWP ₁₀₀
CO ₂	1
R-22	1760
R-134a	1300
R-410A	1924

Hydrofluoroolefins (HFOs)

- Fluorinated propene isomers
 - R-1234yf ($CF_3CF = CH_2$)
 - R-1234ze (CF₃CH = CHF)
- GWP < 4
- Mildly flammable
- Natural Refrigerants



Moving away from Chlorine (ODP) and Fluorine (GWP) inevitably leads to flammability



Goals

Identify appropriate substitute for R-134a as HFCs will phase out:

- Demonstrate an environmentally friendly ENERGY STAR®-qualified residential HFO refrigerant-based HPWH
 - Low GWP, no direct environmental impact
 - No major modification of existing system
 - FHR and EF performance should be comparable



Alternative Refrigerants

Refrigerant	Composition (mass %)				at 155 F			
		T _c (K)	P _c (Mpa)	P _{sat} (Mpa)	h _{fg} (KJ/kg)	P _{vap} (kg/m³)	Vol. Cap (KJ/m³)	P _{sat} (Mpa)
R134a	Pure	374.21	4.06	0.3774	193.17	18.66	3604.55	2.04
R1234yf	Pure	367.85	3.38	0.4006	158.52	22.253	3527.55	1.9725
R1234ze	Pure	382.51	3.64	0.2803	179.49	15.004	2693.07	1.551



Component-Based Flexible Modeling Platform for HPWHs – ORNL Flex HPDM



Model Calibration with Experimental Data

Matching the measured water stratification profile

-Uncalibrated tank model

-Calibrated tank model

Water Draw Patten is based on Pre-2015 EF evaluation criteria

Design Parameters

- 46-gallon water tank
- Heat pump T-stat at the top: on at 115°F, off at 125°F.
- Electric element at the top: on at 110°F, off at 125°F.
- Two different heat loss factors from tank (0.90 and 0.95)
- Two different condenser coil wrap patterns (parallel, counter)
- Two different evaporator sizes and air flow rates (Evap 1 & 2)
- Two different condenser tube sizes (0.31, 0.5 in Nominal)

First Hour Rating (FHR)

Performance Evaluation Criteria

FHR greater or equal to (gals)	FHR less than (gals)	Draw pattern for 24-hr UEF
0	20	Point of use
20	55	Low usage
55	80	Medium usage
80	Мах	High usage

Medium usage draw pattern

Draw Number	Time During Test (hh:mm)	Volume (gals/L)	Flow Rate (GPM/LPM)			
1	00:00	15.0 (56.8)	1.7 (6.5)			
2	00:30	2.0 (7.6)	1 (3.8)			
3	01:40	9.0 (34.1)	1.7 (6.5)			
4	10:30	9.0 (34.1)	1.7 (6.5)			
5	11:30	5.0 (18.9)	1.7 (6.5)			
6	12:00	1.0 (3.8)	1 (3.8)			
7	12:45	1.0 (3.8)	1 (3.8)			
8	12:50	1.0 (3.8)	1 (3.8)			
9	16:00	1.0 (3.8)	1 (3.8)			
10	16:15	2.0 (7.6)	1 (3.8)			
11	16:45	2.0 (7.6)	1.7 (6.5)			
12	17:00	7.0 (26.5)	1.7 (6.5)			
Total Volume Drawn Per Day: 55 gallons (208 L)						

Unified Energy Factor

Coefficient of Performance

Average Supply Water Temperature

Heat Pump Run Time

Total Charge in Both Heat Exchangers

Max. Condenser Sat. Temperature Drop

Max Compressor Discharge Temperature

Average Difference between Supply and **Tank Bulk Water Temperature**

National Laboratory

Conclusions

- R-1234yf can be used as drop-in replacement for R-134a with approximately 2% lower energy factor, and slightly longer heat pump running time due to the reduced capacity.
- R-1234ze performance is comparable as well. However relatively longer HP runtime is disadvantages caused due to relatively lower volumetric capacity of the refrigerant.

Further Developments

Lab testing of the prototype HPWH will confirm the findings and will highlight any potential issues due to the relatively higher flammability of the HFO refrigerants.

Kashif Nawaz

nawazk@ornl.gov

