

Refrigerant-Free Heat Pump Water Heaters Using Solid-State Energy Converters

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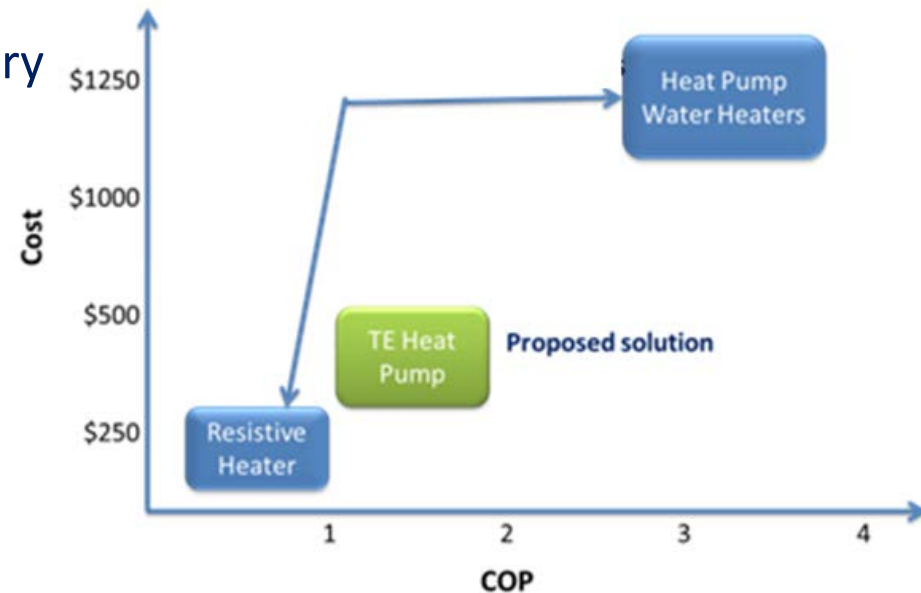
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Rose Metzler

Residential Water Heaters

- 15% of energy consumption. 45% are electric heaters: ~ 1.34 Quads of primary energy
- \$300- \$700 per year energy cost
- Vapor compression based heat pump water heaters are very expensive, and limiting in modularity and flexibility

Source: DOE/EIA

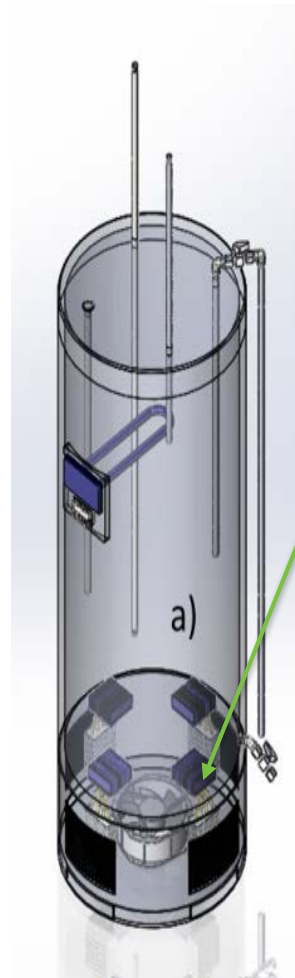
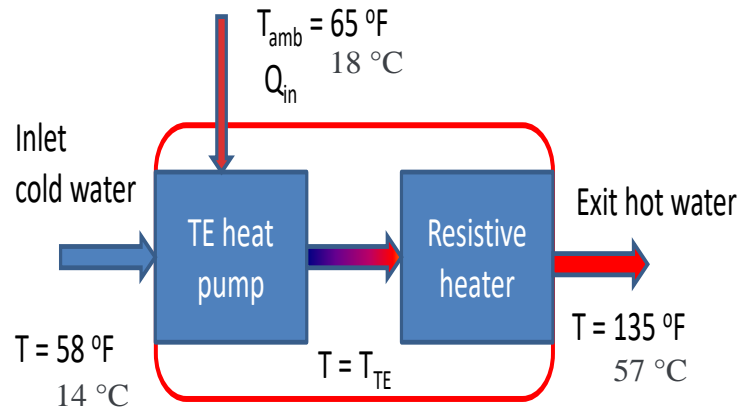


Background

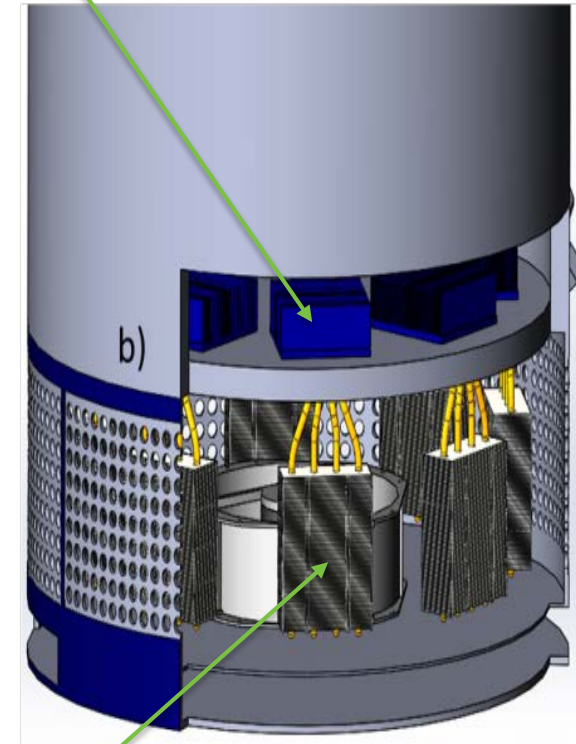
- DOE Program: DE-SC0009231/ Program Manager: Antonio Bouza
- Goal: Demonstrate a home water heater product with affordable and reliable solid-state heat pumps with COP > 1.1 The project includes development of high cooling power thermoelectric modules as well as development of bottom-mount 4-engine and 8-engine heat pumps.

Technical Milestones:

- High Cooling Power Thermoelectric Modules for Heat Pumps
- Bottom Mount 4-Engine Thermoelectric Heat Pump (COP>1.1)
- Plug-In Heat Pump Water Heater (COP>1.5)
- Next Generation Solid State Water Heater (COP>2)

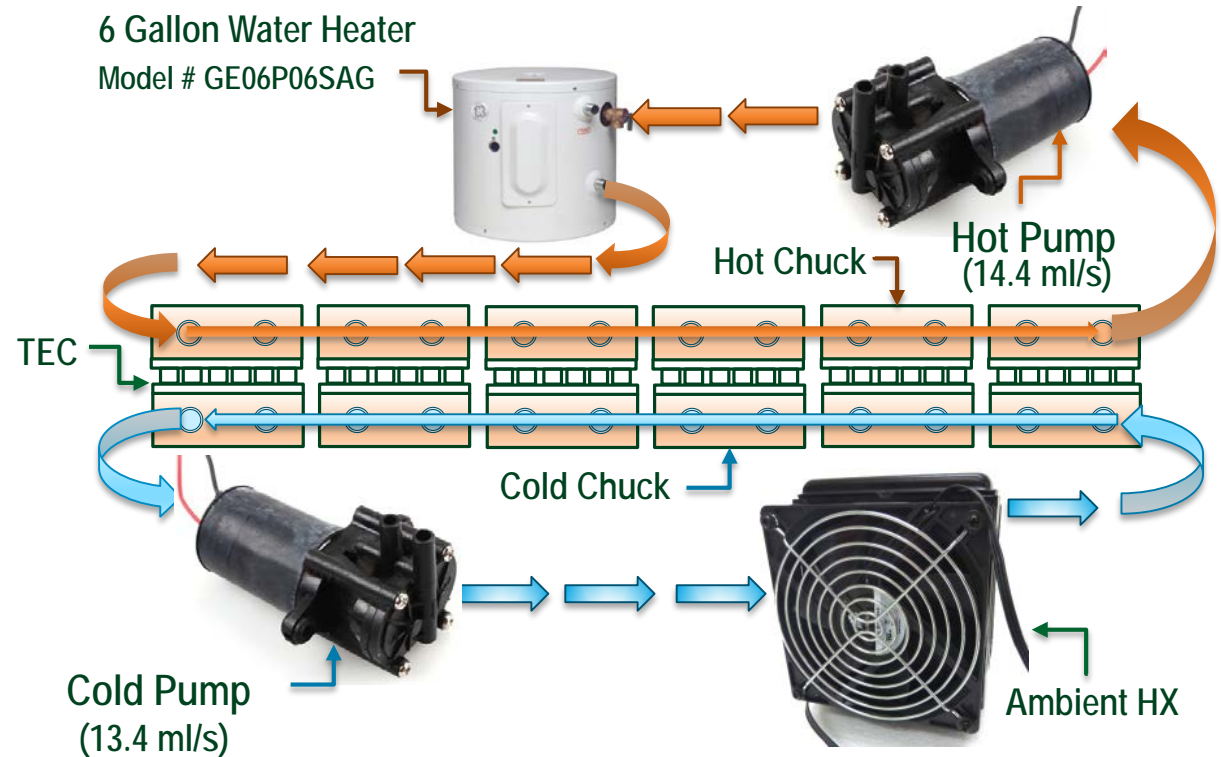
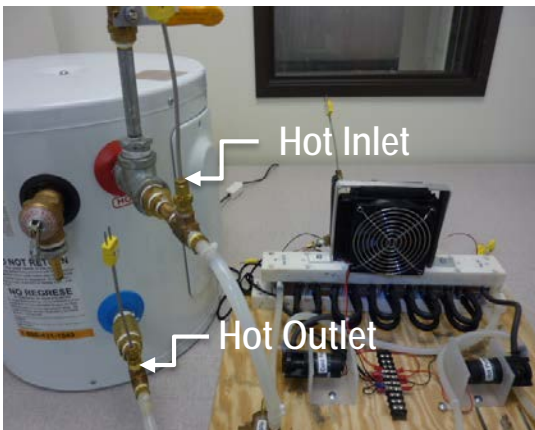
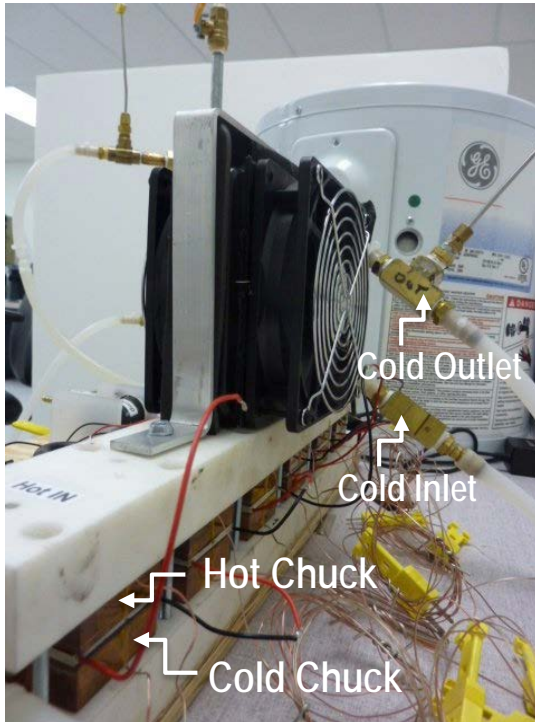


Thermoelectric Hot Side



Thermoelectric Cold Side (Heat-Pipe Heat Sink)

Plug-In TE Heat Pump Water Heater



- Modular, scalable, and compatible
- Addresses needs for capacities 0-100 gallons
- Self controlled loops
- High COP

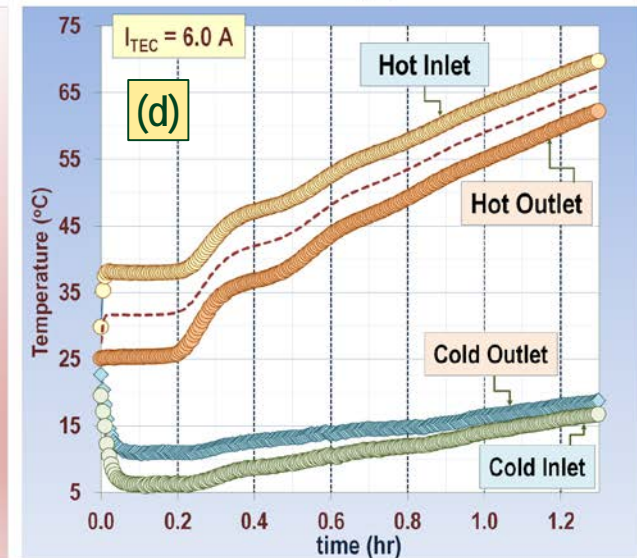
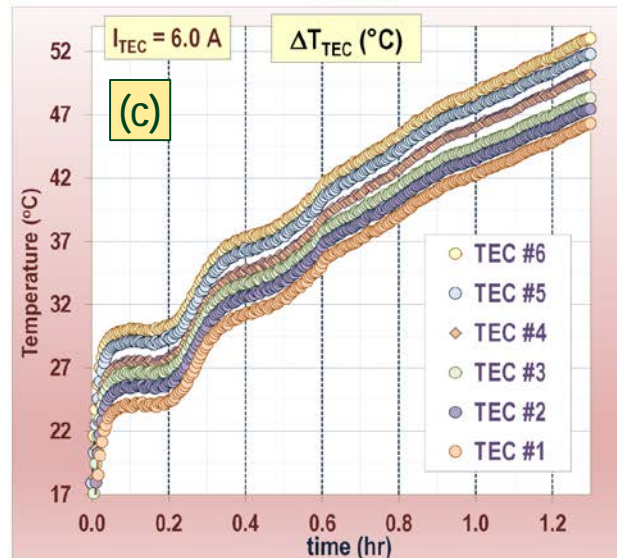
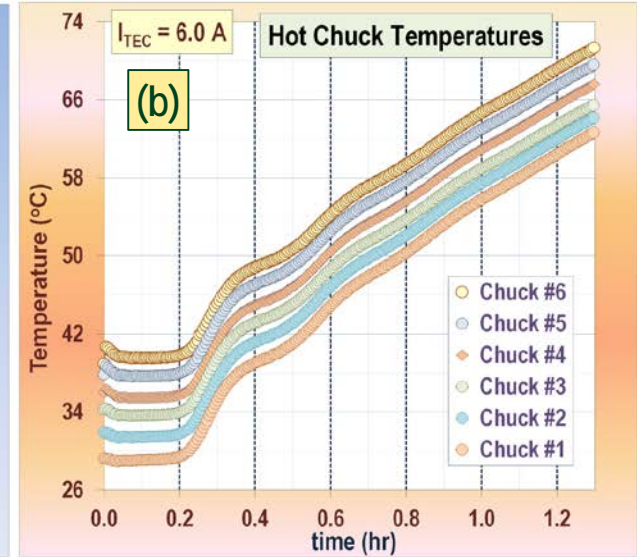
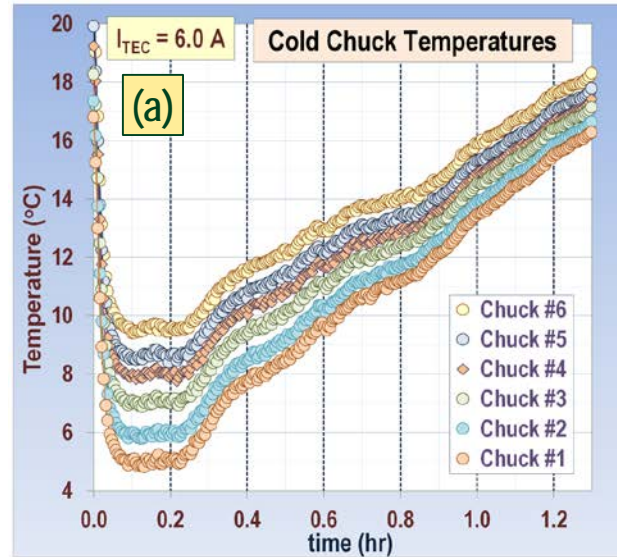
Water Heater Operation for $I_{TEC} = 6A$

Fig. (a) Difference between the 6 Cold Chuck temperatures decreases for $t > 1$ hrs.

Fig. (b) Hot Chuck temperatures (for $t < 0.2$ hours, the thermocline in the container prevent hot temperatures from rising)

Fig. (c) Heat pumping decreases as ΔT_{TEC} increases (@ $t = 1.3$ hrs $\Delta T_{TEC} \sim 46^{\circ}C - 53^{\circ}C$)

Fig. (d) Decrease in heat pumping is indicated as $(T_{cold-outlet} - T_{cold-inlet}) \rightarrow 0$
Average $(T_{hot-inlet}, T_{hot-outlet})$ is indicated by the dotted line.



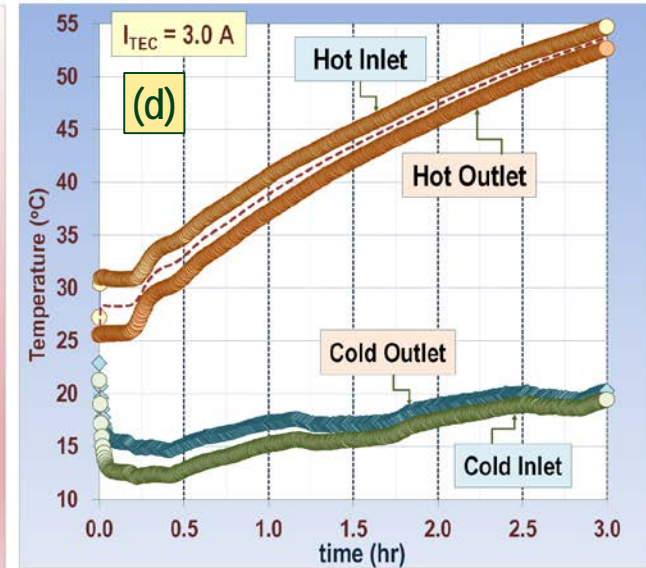
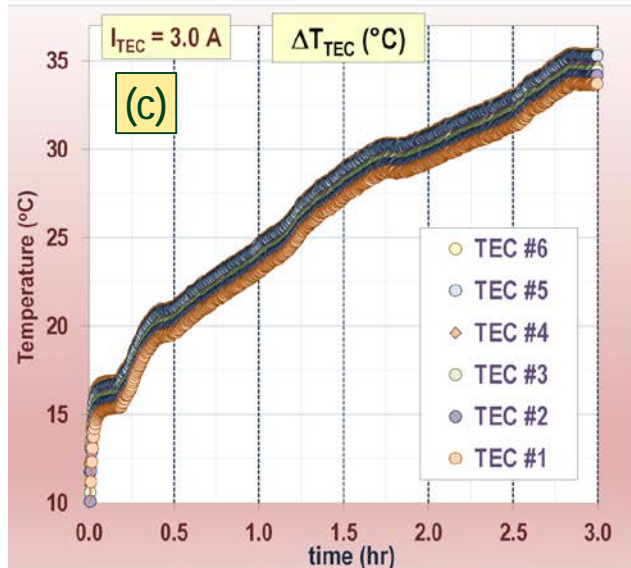
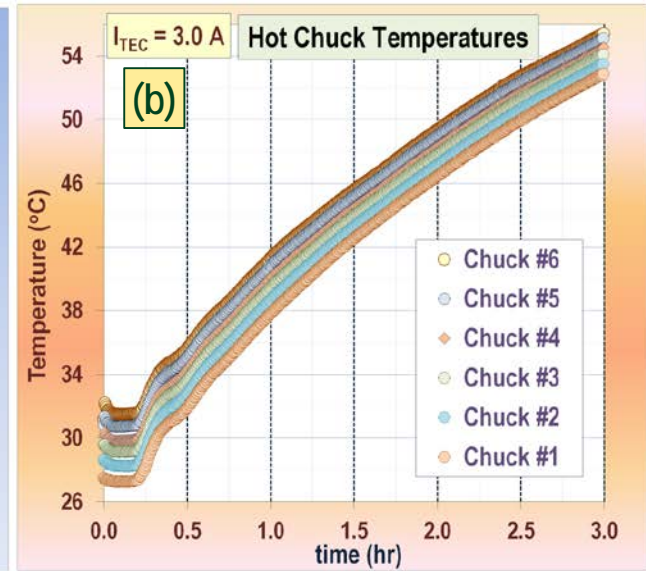
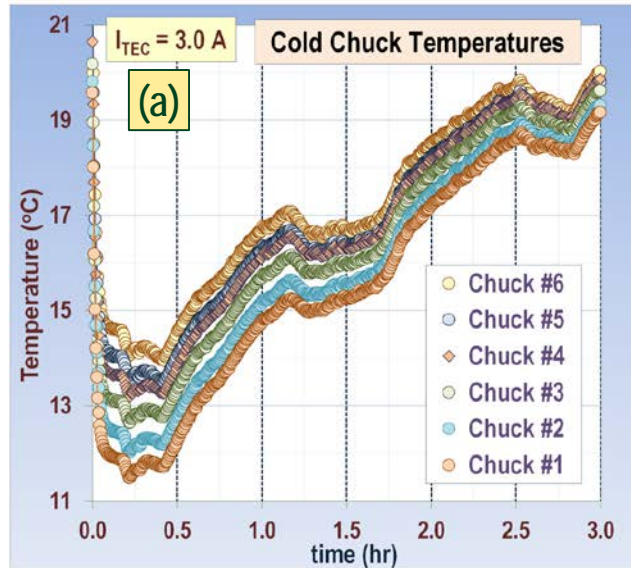
Water Heater Operation for $I_{TEC} = 3A$

Fig. (a) Difference between the 6 Cold Chuck temperatures decreases for $t > 1.5$ hrs.

Fig. (b) Hot Chuck temperatures are closer to each other as compared to $I_{TEC} = 6A$

Fig. (c) Heat pumping $\rightarrow 0$
 $\Delta T_{TEC} \rightarrow \Delta T_{max} (I = 3A)$.
 After 3 hours, $\Delta T_{TEC} \sim 36^{\circ}C - 38^{\circ}C$.

Fig. (d) Decrease in heat pumping is indicated as $(T_{cold-outlet} - T_{cold-inlet}) \rightarrow 0$
 Average $(T_{hot-inlet}, T_{hot-outlet})$ is indicated by the dotted line. $(T_{hot-inlet} - T_{hot-outlet}) \sim 2^{\circ}C$ for $t > 2.5$ hrs

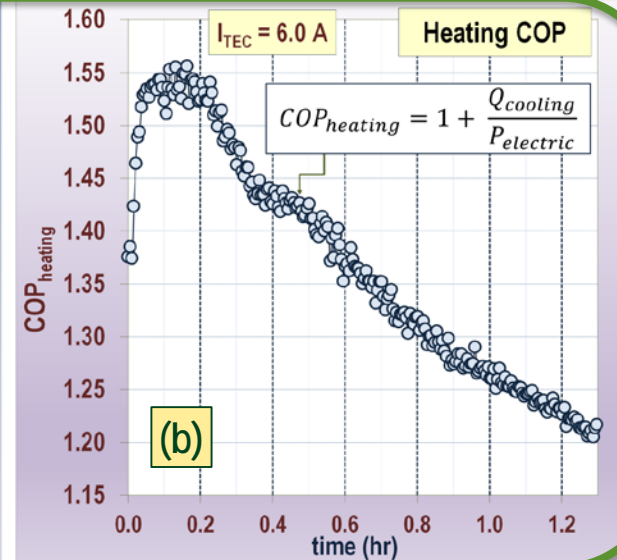
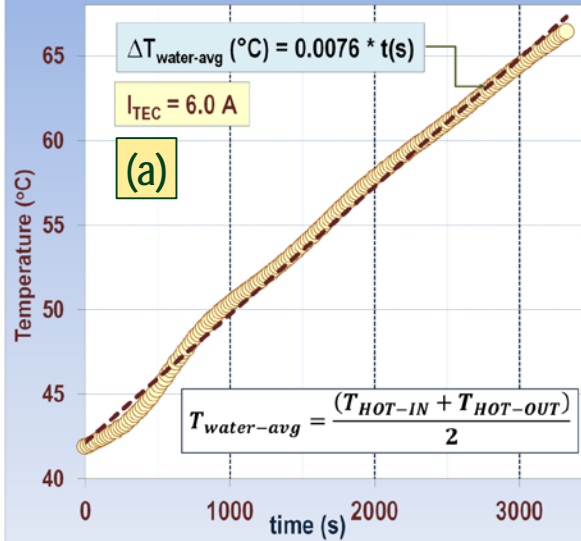


$$I_{TEC} = 6A$$

Fig. (a) COP based on the rate of rise of average hot water temperature

m (kg)	$\delta\Delta T/\delta t$	$Q_{heating}$ (W)	$P_{electric}$ (W)	COP
22.0	0.0076	702	521	1.3

Corroborated with COP estimated from $Q_{cooling}$ in Fig. (b).



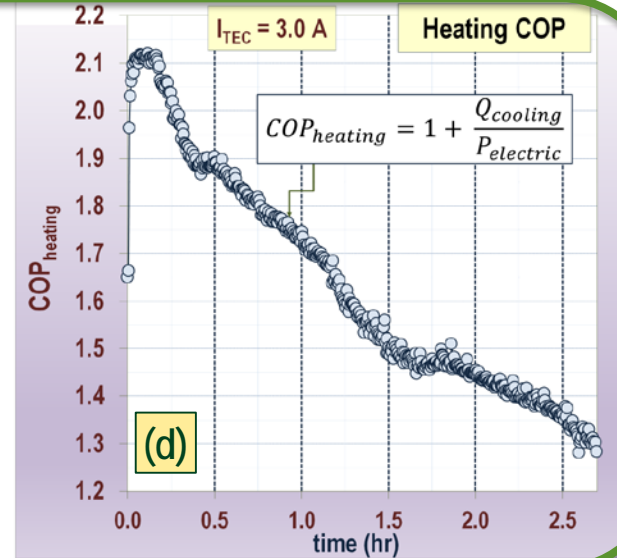
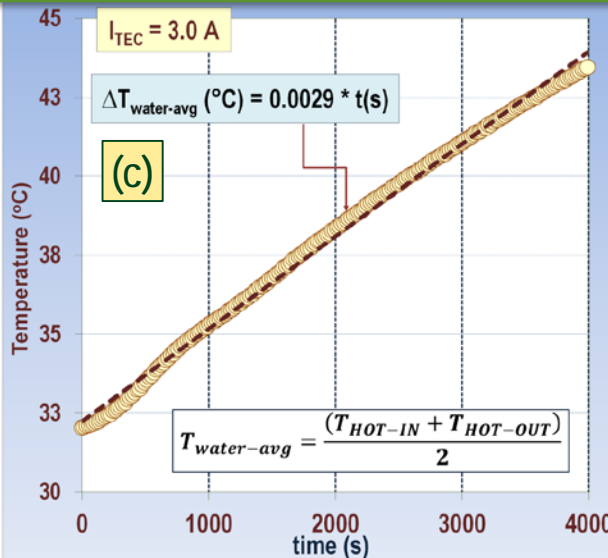
$$I_{TEC} = 3A$$

Fig. (c) COP estimated from the rate of rise of average hot water Temperature

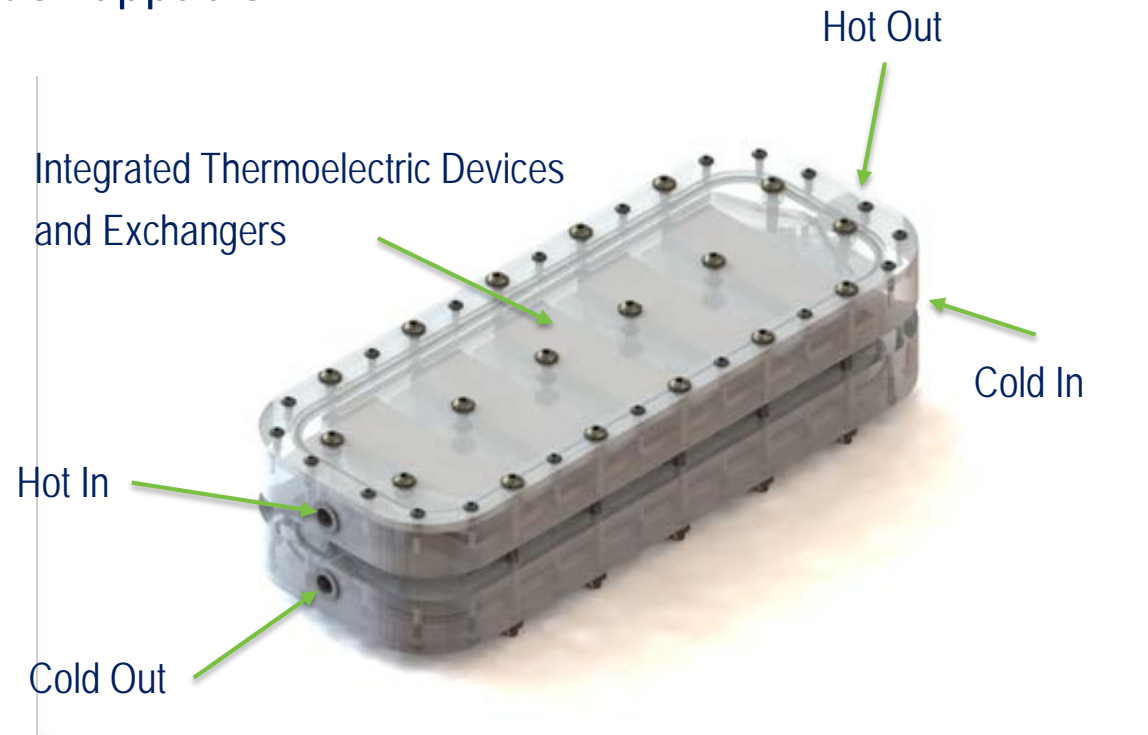
m (kg)	$\delta\Delta T/\delta t$	$Q_{heating}$ (W)	$P_{electric}$ (W)	COP
22.0	0.0029	268	159	1.7

Corroborated with Fig. (d) - COP estimated from $Q_{cooling}$

$$P_{Fan} + P_{pump} = 26.8W$$

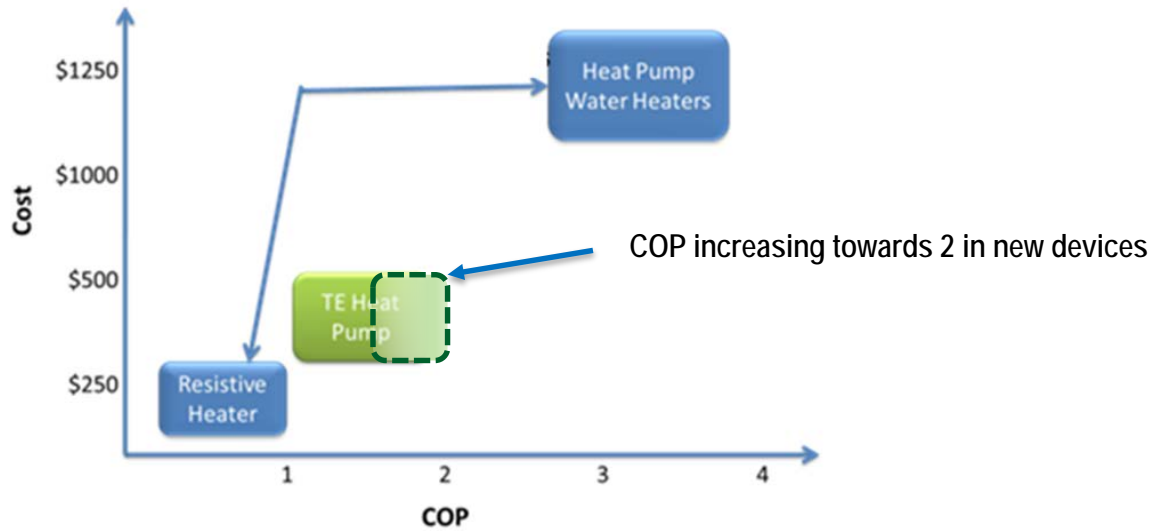


- Multi-Stage TEC Module Design
- Hi Efficiency and Hi Q TEC Devices
- Counter Flow Configuration for High COP
- Plug & Play/Scalable/Hot Swappable

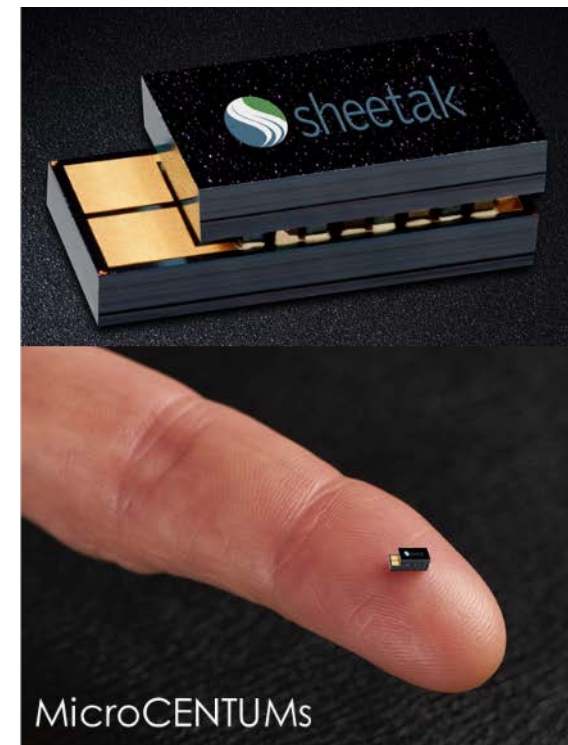


Next Steps and Future Plans

- Continue leading in development and commercialization of efficient thermoelectric heat pumps for refrigeration, water cooling and water heating
- Develop self-controllers for modular heat pumps and extend the plug-in capabilities for variety of cooling/heating applications
- Partner with appliance manufacturers for licensing and sales



- CENTUM[®] High Performance Thermoelectric Devices
 - High heat density due to short transport lengths
 - High efficiency due to higher ZT and nanostructured contacts



Q&A

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